

## Effect of alcohol-based fuel on the performance, combustion and emission characteristics of spark-ignition engine: A review

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### Abstract

The spark ignition (SI) engine is a significant energy device. Alcohol-based alternative fuels have been used to investigate SI engines' performance, combustion, and emission. Using alternative fuels in SI engines, such as alcohol, natural gas, and biodiesel, aims to reduce air pollution and energy costs. Investigations into alternative fuels operating on SI engines have been carried out, and most of the research is focused on performance, combustion, and emission characteristics. This review intends to reveal SI engines' performance, combustion, and emission characteristics using alternative fuels, such as alcohol, natural gas, and biodiesel. The effects of alternative fuels on performance, combustion characteristics, and emissions, such as NO<sub>x</sub>, CO, and HC, were also investigated. The results presented here showed that the use of alcohol fuel blends such as butanol, n-butanol, tert-butanol, iso-butanol, bio-ethanol, isopropanol, propanol, and ethanol-methanol-gasoline can improve engine performance and NO<sub>x</sub>, CO, and HC emissions. Increased performance and emissions have reduced the combustion characteristics. On the contrary, when combustion increases, engine performance and emissions decrease. The engine combustion decreased due to the lower heating value (LHV), lower cetane number, and lower molecular weight of alcohol fuel than gasoline.

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## 1. Introduction

The spark-ignition (SI) engine is a significant energy device for public transportation and private vehicles because it costs less than diesel engines. 30% of the world's greenhouse emissions currently comes from the transportation sector [1–3]. The SI engine is environmentally friendly because fewer emissions come out from the combustion chamber [4–6]. There are several types of substances that are very dangerous in these inhaled suspended particles, such as elemental carbon, inorganic ions, organic carbon and trace elements [7–9]. These chemical compounds can threaten the health and ecosystem [10–15].

The effect of adding iso-butanol-bio-methanol to gasoline can improve the performance of engines such as EGT, BP, Torque and ICP by 0.9%, 2.6%, 1.47% and 6.2% respectively [16]. Meanwhile, the addition of ethanol to gasoline shows a reduction in carbon dioxide (CO<sub>2</sub>), carbon monoxide and nitrogen oxide

(NO<sub>x</sub>) emissions without significantly reducing the power. However, the temperature reduction in the cylinder increases hydrocarbon (HC) emissions [17–20]. NO<sub>x</sub> emissions are scattered in urban environments and mostly come from private vehicles or public transportations which lead to smog, thinning of the ozone layer and acid rain [16,21,22]. Thus, emission reduction from the combustion engine is focused on to reduce environmental and air pollutions [23]. The automotive industry has attempted to devote all its resources with the aim of achieving the requirements for standard emission reductions derived from their vehicles. In this regard, the use of sustainable fuel alternatives such as alcohol, natural gas and biodiesel is the most appropriate method to reduce NO<sub>x</sub>, CO, HC, CO<sub>2</sub> emissions [24,25]. In recent studies, some researchers suggested alternative fuels such as alcohol, natural gas and biodiesel as the main keys to reduce greenhouse gases and harmful pollutants sourced from the combustion chamber [26–31].

Demand for fuel that continues to increase from customers in recent years has resulted in the depletion of fossil fuels [32–34]. Investigations about alternative fuels have been explained in the literature, where reservoirs can meet the demand for fuels such as oil, coal and natural gas [35,36]. The growth of the automotive industry which continues to increase has had an impact on energy scarcity. The increasing demand for alternative fuels and the use of petroleum are the biggest challenges because people want to be ensured of energy security globally. Alternative fuels have been widely used in SI and internal combustion engines with the aim of overcoming the diminishing fossil fuels and rising prices of natural gas. Alcohol fuel, biodiesel and natural gas are alternative fuels that are very promising as a substitute for fossil fuels that have attracted users' interest because they are easily stored and very easy to handle [28,37–40]. SI can be operated by using various mixtures of alcohol fuels with multiple ratios, both through modification and without modification to the engine [17,41–44]. In the blended mode, the alcohol fuel is mixed first with gasoline before being injected into the cylinder [16,21,45–47]. Blended stability with additive fuel is very necessary. The long carbon chain and also the normal HLB (Hydrophile-Lipophile Balance) of 1.6 to 6 contained in fatty alcohols are emulsifiers and co-solvents. The SI engine needs a little modification to add a separate fuel tank, fuel injector, low pressure, the channel and the control system [48–52]. However, in this case, there is no need to use fuel additives because all load conditions and speed of the engine being tested can directly use alcohol fuel [53–57]. Another alternative that can be used for combustion engines as a substitute for fossil fuels is natural gas [49,58–63]. Natural gas has various mixtures of hydrocarbon molecules such as butanol, methanol, ethanol and inert diluents such as nitrogen and carbon dioxide molecules. However, the availability of natural gas throughout the year and geographically varies greatly. In addition, it requires special care during production and operation [24,64–69]. It can be mixed with air to form a homogeneous fuel/air mixture for combustion engines and significantly reduce emissions from the engine exhaust [69–72]. Raw materials such as corn, vegetables, sugar and wheat can be produced into biodiesel fuel [31,49,73–76]. The difference between biodiesel and fossil fuels lies in the oxygen content [77–79]. Clean energy can be produced from biodiesel fuel because it is a renewable energy. Thus, renewable and sustainable fuels such as biodiesel are more promising and also very environmentally friendly [80–82]. This review intends to reveal the performance, combustion and emission characteristics of SI engines using alternative fuels (such as alcohol, natural gas and biodiesel). The effects of alternative fuels on the performance, combustion characteristics and emissions such as NO<sub>x</sub>, CO, HC, and CO<sub>2</sub> were also investigated.

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## **2. Literature review**

### **Alcohol fuel on SI Engine**

Alcohol fuel has been widely studied (especially ethanol and methanol) because it is an alternative fuel that can be mixed into gasoline. Alcohol is a renewable energy source and has oxygen properties that can reduce emissions from the combustion chamber [83,84]. Comparison of alcohol and gasoline is shown in **Table 1**, where alcohol density is higher than that of gasoline. Oxygen, latent heat of evaporation and self-ignition temperature are not present in gasoline; however, the octane number of

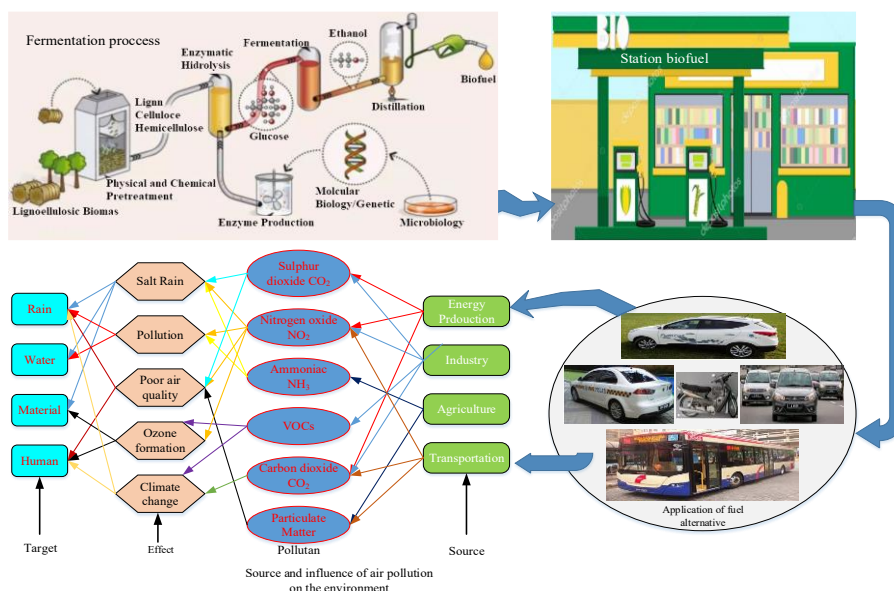
gasolines is higher than alcohol as the number of carbon molecules increases. As an important parameter in engine combustion, low latent heat evaporation and carbon in higher alcohols are not considered. Ethanol and methanol are more desirable than other alcohols as alternative fuels because the resulting emissions are lower and more affordable [85–87]. In a recent study, ethanol-methanol fuel operated on a combustion engine with full load conditions significantly reduced emissions because it has a lower amount of carbon [42,88]. Moreover, the use of ethanol-methanol-propanol, HC and CO emissions was found to decrease because alcohol tolerates higher exhaust-gas recirculation ratio (EGRs) [89–91].

Table 1 - Comparison properties of gasoline and alcohol-based [6,44,46]

| Properties                           | Methanol           | Ethanol                          | Butanol                          | Propanol                         | Gasoline                       |
|--------------------------------------|--------------------|----------------------------------|----------------------------------|----------------------------------|--------------------------------|
| Chemical formula                     | CH <sub>3</sub> OH | C <sub>2</sub> H <sub>5</sub> OH | C <sub>4</sub> H <sub>9</sub> OH | C <sub>3</sub> H <sub>7</sub> OH | C <sub>8</sub> H <sub>15</sub> |
| Cetane number                        | 2                  | 8                                | 17                               | 12                               | 10-15                          |
| Molecular weight (g/mol)             | 32.04              | 46.07                            | -                                | 60.10                            | 96.5                           |
| Density (kg/m <sup>3</sup> )         | 791.3              | 789.4                            | -                                | 803.7                            | 746                            |
| Boiling point (°C)                   | 65                 | 79                               | 117                              | 97                               | 25-215                         |
| Lower heating value (MJ/kg)          | 20.01              | 26.08                            | 32.01                            | 29.82                            | 42.7                           |
| Vaporization latent heat (kJ/kg)     | 1162.64            | 918.42                           | -                                | 727.88                           | -                              |
| Self-ignition temperature (°C)       | 385                | 363                              | -                                | 350                              | -                              |
| Oxygen (%) (wt.)                     | 49.93              | 34.73                            | 21.6                             | 26.62                            | -                              |
| Latent heating (kJ/kg) 25 °C         | 1162               | 904                              | 585                              | 728                              | 380-500                        |
| Viscosity (mm <sup>2</sup> /s) 40 °C | 0.59               | 1.13                             | 2.22                             | 1.74                             | 0.4-0.8 <sup>a</sup>           |

### Natural gas

Natural gas is one of the most important energy sources for use in combustion engines among other alternative fuels [24,92]. There are several advantages to natural gas in terms of the prices that are relatively cheaper and lower greenhouse gas emissions than other alternative fuels. **Table 1** shows the properties found in natural gas. Natural gas fuels can be used in combustion engines with high compression ratios, especially for diesel engines, because the octane is high.



**Figure 1:** Fermentation, station biofuel, application and source and influence of air pollution on the environment

In recent decades, environmental pollution and energy depletion have attracted the attention of governments around the world to use natural gas as an alternative to stationary machines, etc. [93,94]. This is because the induction to the intake or the cylinder directly uses natural gas where fresh air can

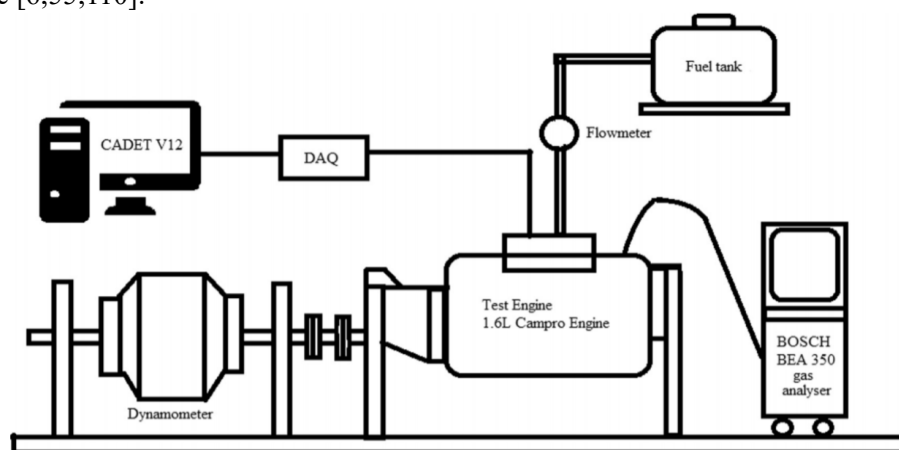
form homogeneous blends, and which are then ignited by a spark plug or fuel pilot [15,95–97]. Therefore, emission reductions derived from reduced exhaust gases and efficient combustion can be achieved. Moreover, modifications do not have to be carried out on vehicles when using natural gas, and thus giving important economic and environmental benefits [98,99].

**Table 2:** Properties of natural gas [6,100]

| Component                      | v/v (%)     |
|--------------------------------|-------------|
| C <sub>5</sub> H <sub>12</sub> | 0.03 ± 0.03 |
| Density(kg/m <sup>3</sup> )    | 0.788       |
| Lower heating value (MJ/kg)    | 49.5 ± 0.2  |
| Stoichiometric air/fuel ratio  | 17.20       |
| CH <sub>4</sub>                | 91.72 ± 1.7 |
| C <sub>3</sub> H <sub>8</sub>  | 1.98 ± 0.8  |
| C <sub>4</sub> H <sub>10</sub> | 0.44 ± 0.5  |
| C <sub>2</sub> H <sub>6</sub>  | 5.5 ± 1.6   |
| CO <sub>2</sub>                | 0.03 ± 0.03 |
| N <sub>2</sub>                 | 0.322 ± 0.3 |

### Biodiesel fuel

Biodiesel alternative fuels are more prominent than others and biodiesel production can be refined from various vegetable oils; for example, cotton seeds, soybeans, sunflowers, peanuts, palm oil and rapeseeds. Biodiesel fuel has been widely used for combustion engines, especially in heavy-duty vehicles and marine engines [101–103]. Table 3 shows the properties of biodiesel fuels. Vegetable oil reacted with alcohol can produce biodiesel and can also be obtained from alkali catalysts (KOH and NaOH) with the transesterification process. This process aims to reduce the viscosity of vegetable oils and oxygen content. The organic substances contained in biodiesel have high molecules such as alcohol, ketones, phenols and ether [104–106]. Biodiesel has a lower density than water because the stability of biodiesel can be stored for a longer duration [55,107–109]. Biodiesel does not have aromatic hydrocarbons and the sulphur content is lower than diesel. Thus, emissions from the production of biodiesel do not endanger human health and are also more environmentally friendly. Moreover, biodiesel has a lower amount of cetane than diesel fuel which further improves engine combustion performance [6,55,110].



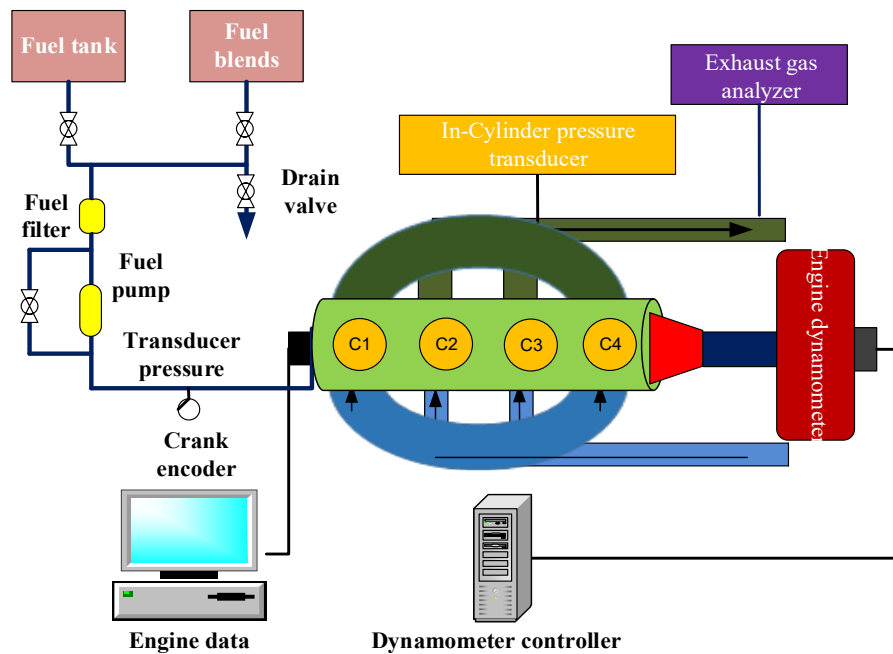
**Figure 2:** Schematic diagram used of the engine test set-up [111]

**Table 3:** Properties of biodiesel fuel

| Properties                | Biodiesel fuel |
|---------------------------|----------------|
| Carbon content (wt%)      | 77.0           |
| Specific gravity at 15 °C | 896            |
| Flash point (°C)          | 105            |

| Properties                    | Biodiesel fuel |
|-------------------------------|----------------|
| Stoichiometric air/fuel ratio | 12.33          |
| Oxygen content (wt%)          | 10.32          |
| Hydrogen content (wt%)        | 12.18          |
| Gross heating value (MJ/kg)   | 40             |
| Cetane number                 | 53             |
| Viscosity at 40 °C (cSt)      | 2.99           |

A schematic diagram for fuel blend modes used by previous researchers is shown in **Figure 2**. This is illustrated from the Campro PROTON engine four-cylinder used to operate alternative fuels. During the experiment, this machine operated at a speed of 1,000 to 5,000 rpm, with 20% butanol and 80% gasoline ratio of fuel. The schematic diagram shown in **Figure 3** is an illustration of the Mitsubishi 4G93 SOHC four-cylinder engine. This machine can be operated using a variety of alternative fuels with speeds of 500 to 6,500 rpm and full engine load conditions.



**Figure 3:** Schematic diagram four-cylinder SI engine

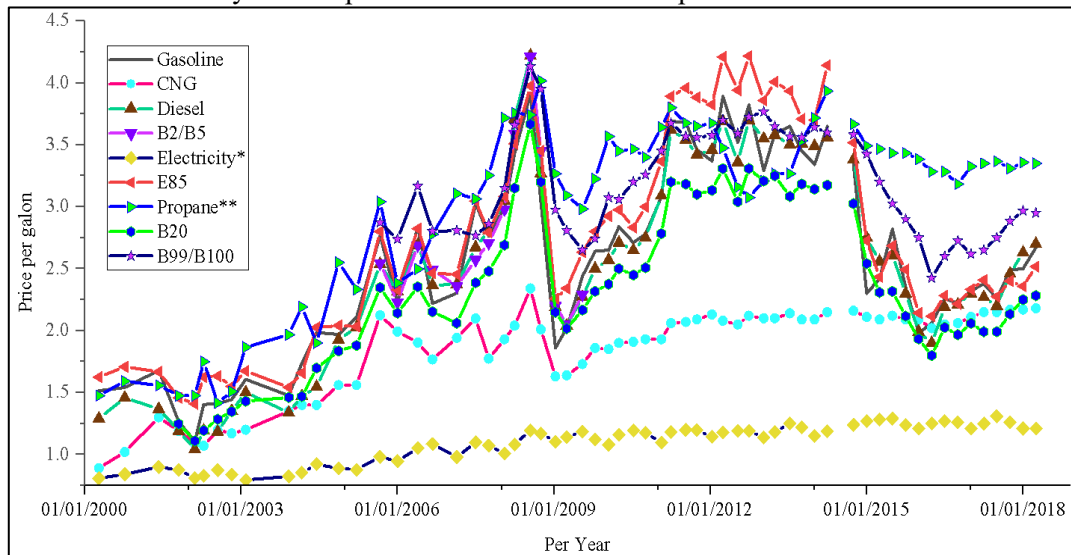
### 3. Methodology

#### Application mode of alcohol fuel on SI Engine

Inside the combustion engine, there are two main application modes used for alternative fuels: combustion mode for mixed fuels and dual fuel combustion mode [112–114]. In this review, the experiments carried out generally used the mixed combustion mode, while alternative fuels used for testing included methanol, ethanol, butanol, propanol, n-butanol, n-propanol, isopropanol, natural gas and biodiesel [115–117]. The purpose of this review is to investigate the effect of using alcohol-based fuels on performance, combustion and emission characteristics of the SI engine. In addition, alternative fuels were used to overcome fossil fuels that are constantly decreasing and prices that are continuously increasing all over the world [118]. The average monthly retail fuel price in the United States (US) from 2000 to 2018 can be seen in the graph shown in **Figure 4**. Gasoline and diesel fuel are the primary drivers for the price of liquid fuels. This is because vehicles that use liquid fuels cannot be replaced and are not dedicated to fuel oil, while electric vehicles are high in cost and not made to reach the lower

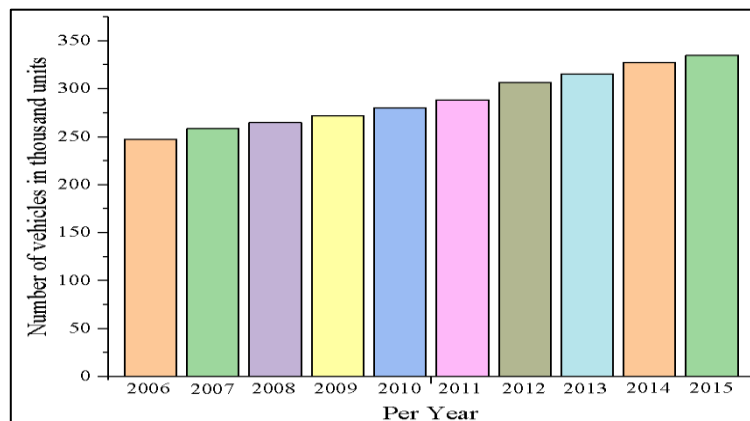


income and middle-class groups of drivers. However, even electricity and natural gas fuels are denied from drivers because only a small portion is marketed for transportation.



**Figure 4:** Alternative fuel vehicles in use, 2000 to 2018

Worldwide commercial vehicle use from 2006 to 2015 is as shown in **Figure 5**. More than 335 million commercial vehicles were used globally in 2015. One of the largest markets using commercial vehicles is in the United States. As a rapid increase in commercial vehicle use also has an impact on fuel, the United States has used alternative fuels to avoid scarcity and increasing demand for fossil fuels.



**Figure 5:** Number of vehicles in thousand units

### Fuel blend mode for engine combustion

Liquid alternative fuels such as alcohol and biodiesel are very suitable using the mixed fuel mode because they can be dissolved into gasoline [24,119,120]. Meanwhile, alcohol-based alternative fuels such as ethanol, methanol, butanol and propanol can be mixed with gasoline with emulsifiers and solvents [47,51]. The schematic diagram of blended fuel mode is shown in **Figure 3**. The phase separation of alternative fuels for engine combustion is a problem faced by the mixed fuel modes. However, this problem can be overcome by using solvents into the mixed fuels. Moreover, some ignition fuels such as methanol can be added to mixed fuels so that the fuel cetane number can be compressed. By realizing this application, the fuel and engine supply systems do not need to be modified because the concentration of these alternative fuels is lower. The use of alternative fuels mixed with gasoline in spark ignition engines and the effects on engine performance, combustion and exhaust gas emissions have been investigated in several previous studies [41,43,62,83,92,121–123]. In this study, the percentage of different mixed fuels was chosen as an experiment. With the addition of alcohol to

gasoline alternative fuels, engine performance and NO<sub>x</sub>, CO and HC emissions increased. However, the combustion characteristics in the engine decreased, while CO<sub>2</sub> emissions were found to increase depending on the concentration of alternative fuel-gasoline used. Moreover, the load and speed of the engine when using alternative-gasoline mixed fuels affected engine performance and NO<sub>x</sub>, CO and HC emissions.

### Mode dual fuel

The dual fuel combustion mode has several advantages and has been extensively studied by previous researchers [86,124–126]. Fig. 2 shows a schematic diagram of a dual fuel combustion mode system. The mixing of alternative fuels was done by inducing into the manifold mixed with fresh air so that a homogeneous mixture can be formed [86]. Furthermore, the alternative fuel mixture was ignited with a spark plug or diesel pilot, when the piston was at the top dead center. The amount of diesel fuel remained constant when the operating load conditions were different, while the amount of alternative fuel injections varied according to the output power of the engine [127]. Alcohol fuel, natural gas and biodiesel were included in the alternative fuels used in the dual fuel combustion mode. In previous research [28,128–133], the use of dual fuel modes was very suitable for alternative alcohol-based fuels such as ethanol as engine combustion fuels. This dual fuel mode can reduce NO<sub>x</sub> emissions compared to using conventional combustion engines. However, dual fuel modes have several difficulties when the combination is performed. The combustion efficiency is much lower when the engine is operated in low and medium engine load conditions. Consequently, it significantly increases NO<sub>x</sub>, CO and HC emissions that do not burn when using a dual fuel combustion mode [134–137]. Therefore, further research is needed to evaluate the effects of engine combustion using dual fuel combustion modes.

## 4. Result & Discussion

### The performance a spark-ignition engine

#### *Brake specific fuel consumption (BSFC)*

The brake specific fuel consumption (BSFC) using alcohol-gasoline mixed fuels requires an increase compared to gasoline for different engine loads and speeds. This shows an increase in BSFC when using alcohol-gasoline mixed fuels compared to pure gasoline. This increase occurred for all engine speeds tested; in the review, it can be reported that increasing BSFC also increased emissions. However, the characteristics of combustion decreased and vice versa; if the combustion increased, this did not apply to BSFC and emissions. The maximum BSFC was obtained from the strength of the brakes when testing the engine using gasoline. Of course, the use of alcohol fuel is more stable because alcohol fuel has a lower energy content than gasoline. Significant increase in BSFC was also due to the increase in engine speed and the results of this review clearly proved the different values of alcohol fuel and gasoline. Comparison of use of alcohol and gasoline is shown in Table 1, and the graph is shown in Table 5.

#### *Brake torque and brake power*

The main performance parameters on the engine are important to be investigated so that performance in the engine can be estimated. These parameters included the brake specific fuel consumption (BSFC), brake power (BP) and engine brake torque (BT). The brake torque in the engine is a rotational force resulting from the cylinder press from the piston crankshaft inside the engine. The average effective cylinder pressure, stroke length and engine load have an impact on engine torque. The engine torque varies greatly when alternative fuels are operated at constant engine loads, due to the nature of the fuel and the effective pressure generated. BSFC calculations can be done with the following equation:

$$BSFC = \frac{\text{Fuel consumption}}{\text{Power output}} \quad (1)$$

Some studies [16,41,111,138,139], stated that the torque curve of alternative fuels has the same results. The effects of using various types of alternative fuels for BSFC, brake power and brake torque are

shown in Table 4. The use of alternative alcohol-based fuels generally improves engine performance. However, when using alternative fuels with a low volume ratio, the engine performance decreases [140]. Increasing the amount of carbon in alcohol fuel has significantly reduced engine losses. The use of alcohol fuel operated in the combustion engine results in increased brake torque, brake power and brake specific fuel consumption [16]. Moreover, the use of alcohol-gasoline fuel with a high ratio resulted in deteriorating engine behaviour. Findings from several previous studies on the performance of machines such as BSFC, BP and BT have no significant impact when using alcohol fuel. However, compared to pure gasoline, alcohol fuel slightly increases engine performance [43,141–143]. As for natural gas, the total variation of BSFC is directly calculated from the ratio between the amount of fuel mass flow rate and engine output power. When the engine is operated in low and medium engine load conditions, engine performance, such as BSFC, is higher. This increase occurs when using the mixed fuel combustion mode compared to conventional combustion mode due to the low combustion temperature and the fuel-air ratio in the combustion chamber which decrease the combustion rate in the engine. However, in conditions with high engine loads, the engine performance decreases for the combustion mode using natural gas fuel on dual fuel modes. The increased combustion temperatures and more mixtures are the two main factors that increase the conversion of natural gas to decrease the total BSFC [24,94,144,145].

**Table 4:** Effect of alcohol on the engine performance.

| Speed<br>RPM | Using fuel   | Volume<br>fraction         | Engine test | BSFC    | Power   | Torque  | Ref.  |
|--------------|--|----------------------------|-------------|---------|---------|---------|-------|
| 1000-5000    | Butanol isomers; n-butanol, sec-butanol, tert-butanol and iso-butanol and gasoline | All fuel 20%               | SI-Gasoline | High ↑  | N/A     | N/A     | [146] |
| 2600-3400    | Iso-butanol–bio-methanol–gasoline and n-butanol–bio-ethanol–gasoline               | 3%, 7% and 10%             | SI-Gasoline | High ↑  | High ↑  | High ↑  | [111] |
| 2600-3400    | n-butanol–methanol–gasoline  | 0%, 3%, 7% and 10%         | SI-Gasoline | N/A     | High ↑  | High ↑  | [16]  |
| 1500-5000    | Ethanol and gasoline   | 10% and 20%                | SI-Gasoline | Lower ↓ | Lower ↓ | N/A     | [147] |
| 1500         | n-Octanol–Diesel   | 10%, 20% and 30%           | SI-Gasoline | Lower ↓ | Lower ↓ | Lower ↓ | [41]  |
| 1000-5000    | Propanol-gasoline  | 0%, 5%, 10%, 15% and 20%   | SI-Gasoline | High ↑  | Lower ↓ | N/A     | [138] |
| 2600-3400    | Bio-ethanol–iso-butanol–gasoline   | 0.8%, 1.2%, 0.4% and 0.6%  | SI-Gasoline | High ↑  | High ↑  | High ↑  | [46]  |
| 1200         | Methanol, ethanol and butanol  | N/A                        | SI-Gasoline | Lower ↓ | High ↑  | High ↑  | [139] |
| 2600-3400    | n-butanol–methanol–gasoline  | 1.5%, 3%, 3.5%, 5% and 7%  | SI-Gasoline | Lower ↓ | Lower ↓ | Lower ↓ | [148] |
| 3000         | iso-octane/n-butanol   | 10%, 20 and 30%            | SI-Gasoline | High ↑  | Lower ↓ | N/A     | [147] |
| N/A          | 1-pentanol   | 5%, 10%, 20%, and 40%      | Kirloskar   | N/A     | N/A     | N/A     | [149] |
| 1200         | Acetone-n-butanol-ethanol and isopropanol-n-butanol-ethanol                        | 30% and 60%                | SI-Gasoline | High ↑  | High ↑  | High ↑  | [150] |
| 1500         | Methanol, ethanol, 2-propanol, 1-butanol and diesel                                | 15%, 30%, 45%, 55% and 70% | CI-engine   | Lower ↓ | Lower ↓ | Lower ↓ | [151] |
| 2000         | Ethanol/gasoline   | 10% and 20%                | SI-Gasoline | High ↑  | High ↑  | High ↑  | [124] |



| Speed<br>RPM | Using fuel                                 | Volume<br>fraction | Engine test | BSFC   | Power  | Torque | Ref.  |
|--------------|--|--------------------|-------------|--------|--------|--------|-------|
| 2000         | isopropanol-n-butanol-ethanol and gasoline | N/A                | SI-Gasoline | High ↑ | High ↑ | High ↑ | [152] |

### Engine combustion characteristics

The effects of combustion such as cylinder pressure and heat release rates from various types of alternative fuels were calculated. The ignition delay is well-defined as the variation of the crank angle in the ignition time and the start of the injection. The zero-braking angle placement was made to determine the ignition time of the second derivative of the cylinder pressure. The collected heat release rate was definite for the crank angle of 90° (CA90), and the variation of the crank angle between CA90 and ignition time was defined for the duration of combustion. The effect of release rate and pressure in the cylinder from the use of alternative alcohol fuels is presented in Table 6. There are many differences in the rate of heat release and in-cylinder pressure from the use of alcohol fuels operated in mixed fuel modes and conventional fuels. Significantly, compression pressure decreased as the alcohol content in the fuel increased, compared to conventional combustion mode [152–155]. This decrease was caused by the latent heat of vaporization in alcohol which is higher than gasoline. So, when intake ports were injected with alcohol fuels such as methanol and ethanol, they absorbed heat and evaporated from the incoming air, so that the cylinder pressure and temperature decreased. When the engine was operated under low load and engine speed conditions, the maximum cylinder pressure changed slightly with the low and medium ratios of premix alcohol; however, the premix was high when the alcohol ratio decreased. Increasing the latent vaporization of alcohol decreased the combustion temperature and cylinder pressure in delaying the excessive combustion phase. The ratio of pre-paste alcohol fuel increased when the engine load and speed were operated. While for natural gas, when operated in a mixed fuel mode with natural gas fuels, a decrease was shown in combustion pressure compared to conventional fuels [156–158]. When the engine was operated with a low load using a mixture of natural gas, the combustion pressure in the cylinder decreased. This was due to the increased specific heat capacity of the cylinder combustion, gas-air mixture leanness and lower ignition delay. At the beginning of combustion, heat cannot be released efficiently because of the amount of steam collected during the ignition; however, after that, it improved. After experiencing an ignition delay, the fuel-air mixture burnt very quickly. Moreover, engine load conditions can cause more accumulation of gas fuel in the combustion chamber and a decrease in ignition delay. The addition of biodiesel can accelerate the initial combustion for all test engine loads because the cylinder pressure increased rapidly in the pressure trace at the end of compression [159,160].

Furthermore, the combustion of the engine with biodiesel mixture was carried out earlier than conventional fuels, especially diesel. The value of the biodiesel fuel mixture occurred a little earlier in the crank angle due to the bulk modulus, higher density and viscosity. By adding a biodiesel mixture, the rate of heat release decreased in the combustion phase and this occurred when the engine was operated in low and high engine load conditions. This decrease also occurred because the injector of the amount of fuel was lower so that the ignition delay was shorter, thus the biodiesel fuel mixture had a low heating value.

### In-Cylinder

Pistons do all the work during compression and produce energy gas through the combustion process in the engine. In an SI engine, when the cylinder gas pressure increases, it can significantly increase combustible fuel. Because the enthalpy of combustion has increased, the fuel energy increases. The combustion characteristics of mixed alcohol fuels can be compared with gasoline fuel through the use of gas pressure in the cylinder. The peak combustion pressure was significantly faster than the gasoline achieved in the cylinder because the octane value in the fuel was lower as shown in the results in Table 5. The higher the octane value, the greater the compression ratio that can be charged in the engine without observing the knock phenomenon in the engine. Thus, a significant increase in engine

performance output can be provided with similar amounts in the use of test fuels. This is a fairly positive aspect, while the engine efficiency is lacking in terms of the current gasoline engine that can be handled.

### Heat release rate

Combustion can produce the rate of heat release in the engine and provide precise information on fuel combustion. The calculation of heat release rate was carried out to try to get information from the combustion process inside the engine. The alcohol fuel mixture was tested to investigate combustion characteristics and compared with gasoline, then compared with the rate of heat release during their combustion process [140]. The increase in heat shown by the heat release pattern can be released by using a gasoline-alcohol fuel mixture. This observation was carried out when the piston has reached the top dead centre. The increase in the amount of heat released in the engine was the result of greater enthalpy in the mixed fuel. Gasoline fuels had a higher level of heat release compared to alcohol-based fuels for all engine speeds and tested loads. This was also found at the location of the peak level of heat release which was wider when using alcohol-gasoline mixed fuels compared to pure gasoline fuel. The main reason in this case was that oxygen contained in alcohol fuel can increase combustion, depending on the concentration given. In addition, combustion using a larger fuel mixture occurred in areas near the dead centre. Therefore, the release of increased heat in the engine can reduce emissions; in the beginning, if the combustion decreased, the emissions increased. This can be influenced by several factors, especially the heat value of gasoline which was higher than alcohol. More details about combustion characteristics are shown in **Table 5**.

**Table 5:** Combustion characteristic

| Speed RPM | Using fuel   | Volume fraction           | Engine test | Pressure | HRR     | Temperature | Ref.  |
|-----------|--|---------------------------|-------------|----------|---------|-------------|-------|
| 1000-5000 | Butanol isomers; n-butanol, sec-butanol, tert-butanol and iso-butanol and gasoline | All fuel 20%              | SI-Gasoline | N/A      | N/A     | High ↑      | [146] |
| 2600-3400 | Iso-butanol–bio-methanol–gasoline and n-butanol–bio-ethanol–gasoline               | 3%, 7% and 10%            | SI-Gasoline | N/A      | N/A     | High ↑      | [111] |
| 2600-3400 | n-butanol–methanol–gasoline  | 0%, 3%, 7% and 10%        | SI-Gasoline | High ↑   | N/A     | High ↑      | [16]  |
| 1500-5000 | Ethanol and gasoline   | 10% and 20%               | SI-Gasoline | Lower ↓  | N/A     | Lower ↓     | [147] |
| 1500      | n-Octanol–Diesel   | 10%, 20% and 30%          | SI-Gasoline | High ↑   | N/A     | N/A         | [41]  |
| 1000-5000 | Propanol-gasoline  | 0%, 5%, 10%, 15% and 20%  | SI-Gasoline | N/A      | High ↑  | High ↑      | [138] |
| 2600-3400 | Bio-ethanol–iso-butanol–gasoline   | 0.8%, 1.2%, 0.4% and 0.6% | SI-Gasoline | N/A      | N/A     | N/A         | [46]  |
| 1200      | Methanol, ethanol and butanol  | N/A                       | SI-Gasoline | Lower ↓  | N/A     | High ↑      | [139] |
| 2600-3400 | n-butanol–methanol–gasoline  | 1.5%, 3%, 3.5%, 5% and 7% | SI-Gasoline | High ↑   | N/A     | N/A         | [148] |
| 3000      | iso-octane/n-butanol   | 10%, 20 and 30%           | SI-Gasoline | High ↑   | N/A     | High ↑      | [147] |
| N/A       | 1-pentanol   | 5%, 10%, 20%, and 40%     | Kirloskar   | High ↑   | N/A     | High ↑      | [149] |
| 1200      | Acetone-n-butanol-ethanol and  | 30% and 60%               | SI-Gasoline | Lower ↓  | Lower ↓ | N/A         | [150] |

| Speed<br>RPM | Using fuel   | Volume<br>fraction         | Engine test | Pressure | HRR    | Temperature | Ref.  |
|--------------|--|----------------------------|-------------|----------|--------|-------------|-------|
| 1500         | isopropanol-n-butanol-ethanol<br>Methanol, ethanol, 2-propanol, 1-butanol and diesel | 15%, 30%, 45%, 55% and 70% | CI-engine   | High ↑   | N/A    | High ↑      | [151] |
| 2000         | Ethanol/gasoline   | 10% and 20%                | SI-Gasoline | High ↑   | High ↑ | High ↑      | [124] |
| 2000         | isopropanol-n-butanol-ethanol and gasoline   | N/A                        | SI-Gasoline | N/A      | N/A    | High ↑      | [152] |

### Engine emission characteristic for alcohol fuel

The use of ethanol led to lower HC and CO emissions from the engine and also decreasing the NO<sub>x</sub> emission. In addition, the blends had lower content of toxicity, since they were reported to be much cleaner after comparing the emissions with the conventional fuels. A detailed literature outcome, depicting the influence of ethanol blends on the various constituents of emissions, has been done in a previous study [35].

### Nitrogen monoxide (NO<sub>x</sub>)

Several studies reviewed in this paper mentioned that NO<sub>x</sub> emissions vary greatly and sometimes increase but reduce combustion in the engine. Conversely, if engine combustion increases and NO<sub>x</sub> emissions decrease, this applies to all engine loads and fuel mixtures. Compared to the alcohol content, the water content of the fuel can significantly reduce emissions; however, it affects the loss of torque from the engine output. Investigations into the use of the fuel mixture have been done. NO<sub>x</sub> emissions do not linearly increased significantly with the compression ratio because the combustion temperature runs throughout the engine. This is because the water content in the fuel has an influence on reducing NO<sub>x</sub> emissions. Furthermore, the peak combustion temperature in the cylinder decreases. As reported, the reaction of nitrogen with engine fuel can form NO<sub>x</sub> emissions. Moreover, it can create greenhouse gases which are very dangerous, because they cannot ignore oxidation from a fuel-rich environment with a lower temperature during the combustion process [17,21,140,161,162].

The exhaust gas temperatures and a comparative graph of various speed and loads of the test engine have provided significant insight into the extent to which the fuel mixture can emit NO<sub>x</sub> emission gas. From several studies reviewed, it was stated that the addition of alcohol fuel to various ratios could reduce NO<sub>x</sub> emissions. However, there are also different studies that stated adding alcohol to gasoline can increase NO<sub>x</sub> emissions. Meanwhile, the amount of gas emitted to gasoline burned enough for certain engine speeds. Conversely, a greater alcohol content in the fuel can produce NO<sub>x</sub> emission levels even at higher exhaust temperatures. This is a determining factor for the level of NO<sub>x</sub> emission stressed from the engine. The increase and decrease of NO<sub>x</sub> emissions in machines that are operated using a mixture of alcohol to gasoline with different engine loads and speeds are shown in Table 6.

The reduction in CO content to zero cannot be done because each combustion is imperfect and there is always a little of the fuel contained in the engine. The concentration of CO emissions in the engine is also affected by the combustion temperature when the engine is operated. In some cases, the amount of CO increases significantly in the exhaust because the combustion temperature in the engine is lower. The changes in the compression ratio in the engine show highly varied emission levels. Every addition of alcohol fuel such as ethanol by 10% to gasoline can reduce CO emissions by 30% [41,163]. In this review, several proven statements that the use of alcohol fuels can reduce CO from various mixtures were tested [16,41,46,111,124,138,139,147]. However, CO emissions also increased in some research, and more details can be shown in Table 6 [115,146–148,150–152]. The CO emission graph clearly illustrated that an increase in alcohol fuel can reduce CO; however, the combustion characteristics decrease slightly and vice versa; if the combustion characteristics of the engine increase, CO emissions decrease. CO emission levels can also be affected by the low latent heat contained in alcohol fuel.

### Hydrocarbon (HC)

HC that are not burned out are hydrocarbons emitted after burning fossil oil in the engine. Some of the air-fuel mixtures inside the engine piston is hiding from the combustion of fire in the cracks available on the piston ring grooves. Moreover, it is possible that from some combustion chambers, the flames are very weak, and the combustion temperatures are low. As a result, when the fuel is not burned, the steam emitted from HC is formed. The HC emission from alcohol fuel is larger compared to gasoline. This is because the calorific value and molecular weight in alcohol fuel are lower than gasoline [6,44,46]. The effect of an alcohol-based alternative fuel mixture on hydrocarbon emissions is significantly increased. However, from some previous studies, not all HC experienced an increase, as shown in **Table 6**. Non-combustible HC emissions are one of the emissions in the engine discussed in this review. The engine efficiency decreases with the increasing HC emissions; on the contrary, if engine combustion increases, HC emissions decrease. The wasteful use of fuel in the engine also causes an increase in pollution levels. In the case of a fuel mixture, it can provide better combustion and is quite clear because it can limit the unburned fuel trapped in a particular engine location. Alcohol fuels with B20 and B40 fuel mixtures increase HC emissions due to the lower cetane number which causes ignition delays [150,152]. Meanwhile, a mixture of gasoline-ethanol and methanol HC emission was found to decrease, while the HC methanol-gasoline mixture increased. The increasing HC uses methanol-gasoline because too much fuel is injected, resulting in ignition delays [148]. The addition of ethanol fuel with a mixture ratio of E10W and E20W to gasoline reduces HC emissions compared to E20W, E10W [41]. The effect of iso-butanol-n-butanol-gasoline mixture produced the lowest HC emissions compared to pure gasoline when the engine load was high [111]. The effect of oxygenated fuel tested on dual fuels showed that HC emissions were increased compared to conventional fuel modes [124]. An alcohol fuel mixture of 10% (IBE10%) increased UHC emissions by 12.4-25.1 and 4.4-6.1% compared to ABE. However, deteriorating combustion quality can increase UHC emissions [151]. HC emissions decreased by 14.18% from the use of propanol-gasoline fuel mixture. In addition, vehicle fuel consumption increased with decreasing hydrocarbons [46].

**Table 6:** Emission Characteristic

| Speed RPM | Using fuel   | Volume fraction           | Engine test | NOx     | CO      | HC      | CO <sub>2</sub> | Ref.  |
|-----------|--|---------------------------|-------------|---------|---------|---------|-----------------|-------|
| 1000-5000 | Butanol isomers; n-butanol, sec-butanol, tert-butanol and iso-butanol and gasoline | All fuel 20%              | SI-Gasoline | High ↑  | High ↑  | High ↑  | N/A             | [146] |
| 2600-3400 | Iso-butanol-bio-methanol-gasoline and n-butanol-bio-ethanol-gasoline               | 3%, 7% and 10%            | SI-Gasoline | Lower ↓ | Lower ↓ | Lower ↓ | High ↑          | [111] |
| 2600-3400 | n-butanol-methanol-gasoline  | 0%, 3%, 7% and 10%        | SI-Gasoline | N/A     | Lower ↓ | Lower ↓ | Lower ↓         | [16]  |
| 1500-5000 | Ethanol and gasoline   | 10% and 20%               | SI-Gasoline | N/A     | High ↑  | High ↑  | Lower ↓         | [147] |
| 1500      | n-Octanol-Diesel   | 10%, 20% and 30%          | SI-Gasoline | Lower ↓ | Lower ↓ | Lower ↓ | N/A             | [41]  |
| 1000-5000 | Propanol-gasoline  | 0%, 5%, 10%, 15% and 20%  | SI-Gasoline | Lower ↓ | Lower ↓ | Lower ↓ | N/A             | [138] |
| 2600-3400 | Bio-ethanol-iso-butanol-gasoline   | 0.8%, 1.2%, 0.4% and 0.6% | SI-Gasoline | High ↑  | Lower ↓ | Lower ↓ | N/A             | [46]  |
| 1200      | Methanol, ethanol and butanol  | N/A                       | SI-Gasoline | N/A     | Lower ↓ | Lower ↓ | High ↑          | [139] |
| 2600-3400 | n-butanol-methanol-gasoline  | 1.5%, 3%, 3.5%, 5% and 7% | SI-Gasoline | Lower ↓ | High ↑  | Lower ↓ |                 | [148] |
| 3000      | iso-octane/n-butanol   | 10%, 20 and 30%           | SI-Gasoline | N/A     | Lower ↓ | Lower ↓ | High ↑          | [147] |

| Speed<br>RPM | Using fuel   | Volume<br>fraction            | Engine<br>test  | NOx    | CO      | HC     | CO <sub>2</sub> | Ref.  |
|--------------|--|-------------------------------|-----------------|--------|---------|--------|-----------------|-------|
| N/A          | 1-pentanol   | 5%, 10%, 20%,<br>and 40%      | Kirloskar       | N/A    | N/A     | N/A    | N/A             | [149] |
| 1200         | Acetone-n-butanol-<br>ethanol and<br>isopropanol-n-<br>butanol-ethanol | 30% and 60%                   | SI-<br>Gasoline | High ↑ | High ↑  | High ↑ | N/A             | [150] |
| 1500         | Methanol, ethanol, 2-<br>propanol, 1-butanol<br>and diesel             | 15%, 30%, 45%,<br>55% and 70% | CI-<br>engine   | High ↑ | High ↑  | High ↑ | N/A             | [151] |
| 2000         | Ethanol/gasoline   | 10% and 20%                   | SI-<br>Gasoline | High ↑ | Lower ↓ | High ↑ | Lower<br>↓      | [124] |
| 2000         | isopropanol-n-<br>butanol-ethanol and<br>gasoline                      | N/A                           | SI-<br>Gasoline | High ↑ | High ↑  | High ↑ | N/A             | [153] |

The use of alcohol-based fuel with fumigation and mixtures in combustion engines has been widely investigated. Alcohol fuels, such as methanol, ethanol, butanol and biodiesel, have several advantages. Alcohol fuel is suitable for cleaning machines such as SI engines and the like, both through fumigation and mixture into conventional ingredients. The disadvantages of using alcohol fuels such as methanol are mainly for diesel engines because of lower calorific value, low flash point and smaller cetane number. The advantages and disadvantages of alcohol-based fuels are described in **Table 7** [31,42]. Among several alternative fuels used for combustion engines, biodiesel has more advantages that can support some unique qualities and features. Biodiesel fuel has been tested, and all the requirements for health effects have been successfully passed, unlike other alternative fuels, as standardized by the Amendment to the Clean Water Act in 1990.

**Table 7:** Comparative study of alcohol, natural gas and biodiesel

|                                   |   |
|-----------------------------------|---|
| <b>Advantages<br/>Methanol</b>    | <ul style="list-style-type: none"> <li>• Higher cooling, hence less compression work</li> <li>• The high latent heat of vaporisation</li> <li>• The high stoichiometric fuel-air ratio</li> <li>• The high stoichiometric fuel-air ratio</li> <li>• High oxygen content, high hydrogen to the carbon ratio and low sulfur content</li> <li>• Reduced soot and smoke</li> </ul>  |
| <b>Disadvantages<br/>Methanol</b> | <ul style="list-style-type: none"> <li>• Poor combustion characteristics</li> <li>• Lower energy content</li> <li>• Longer ignition delay</li> <li>• More corrosive</li> <li>• Lower flash point</li> </ul>   |
| <b>Advantages<br/>biodiesel</b>   | <ul style="list-style-type: none"> <li>• Biodiesel fuel is a renewable energy source unlike petroleum-based diesel</li> <li>• One of the main biodiesel fuel advantages is that it is less polluting than petroleum diesel</li> <li>• Excessive production of soybeans in the world makes it an economical way to utilise this surplus for manufacturing the Biodiesel fuel</li> <li>• The lack of sulfur in 100% biodiesel extends the life of catalytic converters</li> <li>• Another of the advantages of biodiesel fuel is that it can also be blended with other energy resources and oil</li> <li>• It can also be distributed through existing diesel fuel pumps, which is another biodiesel fuel advantage over other alternative fuels</li> <li>• The lubricating property of the biodiesel may lengthen the lifetime of engines</li> <li>• Biodiesel fuel can also be used in existing oil heating systems and diesel engines without making any alterations</li> </ul> |



|                                   |   |
|-----------------------------------|---|
| <b>Disadvantages of biodiesel</b> | <ul style="list-style-type: none"> <li>• It requires energy to produce biodiesel fuel from soy crops, plus there is the energy of sowing, fertilizing and harvesting</li> <li>• At present, Biodiesel fuel is about one and a half times more expensive than petroleum diesel fuel</li> <li>• Another biodiesel fuel disadvantage is that it can harm rubber hoses in some engines</li> <li>• Biodiesel fuel distribution infrastructure needs improvement, which is another of the biodiesel fuel disadvantages</li> <li>• As Biodiesel cleans the dirt from the engine, this dirt can then get collected in the fuel filter, thus clogging it. So, filters have to be changed after the first several hours of biodiesel use.</li> </ul>    |
| <b>Advantages Ethanol</b>         | <ul style="list-style-type: none"> <li>• The use of ethanol as opposed to petroleum could reduce carbon dioxide emissions, provided that a renewable energy resource was used to produce crops required to obtain ethanol and to distil fermented ethanol</li> <li>• Unlike petroleum, ethanol is a renewable resource</li> <li>• Ethanol burns more cleanly in the air than petroleum, producing less carbon (soot) and carbon monoxide</li> </ul>   |
| <b>Disadvantages Ethanol</b>      | <ul style="list-style-type: none"> <li>• Large amounts of arable land are needed to produce the necessary plants to produce ethanol, leading to problems such as soil erosion, deforestation, fertiliser run-off and salinity</li> <li>• Typical current engines would require modification to use high concentrations of ethanol</li> <li>• Ethanol has a lower heat of combustion (per mole, per unit of volume, and per unit of mass) than petroleum</li> <li>• Major environmental problems would arise out of the disposal of waste fermentation liquors.</li> </ul>   |
| <b>Advantages Butanol</b>         | <ul style="list-style-type: none"> <li>• Biomass is readily available for making butanol, and the process is simple enough that it can be wholly regionalized (from biomass production to fuel).</li> <li>• Butanol is much like gasoline so that it can be distributed through the same infrastructure, and it's much more comfortable on the atmosphere and planet than is petroleum-based fuel.</li> <li>• Butanol is very water tolerant, so it can mix easily with gasoline or diesel without the separation issues that water can bring to ethanol mixtures.</li> <li>• It can also be burned in most combustion engines made for it without the need of water separation (as with diesel) or dehydration (as with ethanol).</li> </ul> |
| <b>Disadvantages Butanol</b>      | <ul style="list-style-type: none"> <li>• Butanol has less energy density than gasoline and a lower octane rating than gas, ethanol, or methanol.</li> <li>• It has an air-fuel ratio similar to gasoline (11.1 vs 14.6) but outputs less energy overall.</li> <li>• While the making of butanol is not new, producing it correctly as a fuel is, so no real commercialization has happened yet.</li> </ul>  |

## 5. Conclusion

This review presents the effects of alcohol mixture on the performance, combustion and emission characteristics on spark ignition (SI) engines. The addition of alcohol fuel to gasoline can result in more varied engine operations. The increased combustion has an impact on the reduction in performance and emissions in the engine. The conclusions that can be drawn from this paper are as follows:

- The fuel with more octane numbers has far better implications for engine efficiency; more octane contained in the fuel can provide better engine combustion.
- The percentage of alcohol fuel increases NO<sub>x</sub>, CO, HC emissions and performance in the engine. However, it reduces the combustion characteristics; otherwise, if the combustion in the engine increases, emissions and engine performance decrease.
- The use of new and clean alternative fuels in engines such as alcohol-based fuels can offer longer injection delay, shorter ignition delays and lower pressure ratios. The properties offered are contrary to what is found in conventional fuels.

- Clean alternative fuels for machines such as alcohol, biodiesel and natural gas have enormous potential in the future to overcome the depletion of energy worldwide.

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| Nomenclature                     |   |                                  |                                      |
|----------------------------------|---|----------------------------------|--------------------------------------|
| BSFC                             | Brake Specific Fuel Consumption         | LHV                              | Low Heat Value                       |
| BTE                              | Brake Thermal Efficiency                | BMEP                             | Brake Mean Effective Pressure        |
| HCCI                             | Homogeneous Charge Compression Ignition | PCCI                             | Premixed Charge Compression Ignition |
| E                                | Engine's speed surface                  | PAH                              | Polycyclic Aromatic Hydrocarbon      |
| NO <sub>x</sub>                  | Nitrogen Oxides                         | HC                               | Hydrocarbon                          |
| CO                               | Carbon monoxide                         | PM                               | Particulate Matter                   |
| HLB                              | Hydrophile Lipophile Balance            | EGR <sub>s</sub>                 | Exhaust Gas Recirculation            |
| BP                               | Brake power                             | BT                               | Brake torque                         |
| SI                               | Spark ignition                          | CO <sub>2</sub>                  | Carbon dioxide                       |
| HRR                              | Heat release rate                       | CH <sub>3</sub> OH               | Methanol                             |
| C <sub>2</sub> H <sub>5</sub> OH | Ethanol                                 | C <sub>3</sub> H <sub>7</sub> OH | Propanol                             |
| C <sub>4</sub> H <sub>9</sub> OH | Butanol                                 | C <sub>8</sub> H <sub>15</sub>   | Gasoline                             |

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